

Fluids: Introduction and Potential Flow

Euler Equations and Ideal Fluid Mechanics

Basic Eqs.

(i) Euler Eqs.

ideal - 'dry' H_2O

- no viscosity, molecular diff.

a.) Continuity Equation

Consider $\left\{ \begin{array}{l} \text{Volume } V \text{ of fluid} \\ \text{density } \rho(\underline{x}, t) \end{array} \right.$



Then, mass conservation \Rightarrow

$$\frac{dM}{dt} = \frac{\partial}{\partial t} \int dV \rho(\underline{x}, t) = - \int d\underline{s} \cdot [\rho(\underline{x}, t) \underline{v}(\underline{x}, t)]$$

\downarrow net change in mass \downarrow mass density flux
 \downarrow net out/in flow

$$\frac{\partial}{\partial t} \int dV \rho(\underline{x}, t) = - \int dV \nabla \cdot (\rho \underline{v})$$

$$\boxed{\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \underline{v}) = 0}$$

Continuity Eqn.
(non/linear)

$\rho \underline{v} \equiv$ mass density flux

$$\frac{\partial \rho}{\partial t} + \underbrace{\underline{v} \cdot \nabla \rho}_{\text{advection}} = -\rho \underbrace{\nabla \cdot \underline{v}}_{\text{compression}}$$

b.) Momentum Balance Equation

Consider force acting on fluid element of density $\rho(\underline{x}, t)$ and velocity $\underline{v}(\underline{x}, t)$

$$\rho \underline{a} = \underline{f}$$

↳ force density

$$\underline{f} = -\nabla p + \underline{f}_{\text{ext}}$$

↓
 pressure gradient (fluid on self) ↳ external force
 c.i.e. $\underline{f}_{\text{ext}} = \rho \underline{g}$

As to \underline{a} , $\underline{v} = \underline{v}(\underline{x}, t)$, so need consider:

- change in velocity (i.e. acceleration) at fixed point

$$\text{i.e. } \underline{a} = \frac{\partial \underline{v}}{\partial t}$$

- rate of change of momentum of moving fluid particle (i.e. moving in space)

then

$$\underline{dV} = \left(\frac{\partial \underline{V}}{\partial t} \right) dt + \underline{dr} \cdot \underline{\nabla} \underline{V}$$

\downarrow local variation \downarrow particle moves in spatially dependent velocity field.

so

$$\frac{d\underline{V}}{dt} = \frac{\partial \underline{V}}{\partial t} + \underline{V} \cdot \underline{\nabla} \underline{V}$$

\downarrow convective, advective, substantive derivative

so

$$\frac{d\underline{x}}{dt} = \underline{V}(\underline{x}, t)$$

$$\rho \left(\frac{\partial \underline{v}}{\partial t} + \underline{v} \cdot \nabla \underline{v} \right) = -\nabla p + \underline{f} \quad \left\{ \begin{array}{l} \text{Euler Eqn.} \\ \text{(momentum balance,} \\ \text{non/linear)} \end{array} \right.$$

Useful to note for momentum flux;

$\rho \underline{v} \equiv$ fluid momentum-per-volume $\left\{ \begin{array}{l} \text{combine} \\ \text{continuity,} \\ \text{Euler} \end{array} \right.$

$$\frac{\partial (\rho \underline{v})}{\partial t} = \underline{v} \frac{\partial \rho}{\partial t} + \rho \frac{\partial \underline{v}}{\partial t}$$

$$= -\underline{v} \left(\rho (\nabla \cdot \underline{v}) + \underline{v} \cdot \nabla \rho \right) + \rho \left(-\underline{v} \cdot \nabla \underline{v} - \frac{\nabla p}{\rho} \right)$$

$$= - \left(\rho \left[\underline{v} (\nabla \cdot \underline{v}) + \underline{v} \cdot \nabla \underline{v} \right] + \underline{v} (\underline{v} \cdot \nabla \rho) \right) - \nabla p$$

$$= - \nabla \cdot \left(\rho \underline{v} \underline{v} + \underline{I} p \right)$$

\downarrow Reynolds stress tensor
 (analogous Maxwell stress tensor)

\downarrow identity tensor

\rightarrow momentum flux tensor

\rightarrow pressure

$$\frac{\partial (\rho v_i)}{\partial t} = - \frac{\partial}{\partial x_k} \Pi_{ik}$$

dynamic pressure

(thermal) pressure

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$$\Pi_{ik} = \rho v_i v_k + \delta_{ik} p \quad \Rightarrow \text{Defined momentum flux}$$

i.e. $\rho \underline{v} \equiv$ momentum density

$$\begin{aligned} \Rightarrow \frac{dI}{dt} &= \frac{d}{dt} \int dV \rho \underline{v} = - \int dV \underline{\nabla} \cdot (\rho \underline{v} \underline{v} + \underline{\underline{I}} p) \\ &= - \int d\underline{S} \cdot (\rho \underline{v} \underline{v} + \underline{\underline{I}} p) \end{aligned}$$

↓
defines momentum outflow

$\therefore \Pi_{ik} dS_k \equiv$ momentum flux in i^{th} direction

c.) Energy Equation - Ideal Fluid

In ideal fluid, no heat is exchanged between fluid elements \Rightarrow motion adiabatic

$S \equiv$ entropy per mass

↑
beware terminology

$$\Rightarrow \frac{dS}{dt} = 0$$

$$\frac{\partial S}{\partial t} + \underline{v} \cdot \underline{\nabla} S = 0$$

adiabatic eqn for fluid.

$$\rho \frac{Dp}{Dt} + \nabla \cdot (\rho \underline{v}) = 0$$

$$\rho \frac{DS}{Dt} + \nabla \cdot (\rho S \underline{v}) = 0$$

$$\Rightarrow \boxed{\frac{D}{Dt} (\rho S) + \nabla \cdot (\rho S \underline{v}) = 0} \quad (\text{Egn. Continuity for entropy})$$

$\rho S \underline{v} \equiv$ entropy flux

Applications - Euler Egn.

Now, further develop a.) - Bernoulli Egn.
b.) - Energy Flux } topics in ideal fluids
c.) - Kelvin Thm

a.) Bernoulli Egn.

Useful to note for isentropic fluid (constant entropy)

$$\Rightarrow S = \text{const.}$$

$$\text{out } dW = T ds + v dp$$

↓
specific enthalpy

$$\begin{aligned} dE &= dQ - p dV \\ &= T ds - p dV \\ dW &= T ds + v dp \end{aligned}$$

↳ $v \equiv$ specific volume

e. $dE = dQ - pdV$
 $= Tds - pdv$

$W = E + PV$ (enthalpy is Legendre transform of energy)
enthalpy

Const. entropy $\Rightarrow dw = vdp$
 $= \frac{dp}{\rho}$

\Rightarrow can re-write Euler equation as:

$\frac{\partial \underline{v}}{\partial t} + \underline{v} \cdot \nabla \underline{v} = -\nabla W$ { Euler Eqn. - Isentropic fluid

Now, $\underline{v} \cdot \nabla \underline{v} = -\underline{v} \times (\nabla \times \underline{v}) + \frac{1}{2} \nabla (v^2)$

$\nabla \times \underline{v} = \underline{\omega} \equiv \text{vorticity}$

$\Rightarrow \frac{\partial \underline{v}}{\partial t} = \underline{v} \times \underline{\omega} - \nabla (W + \frac{v^2}{2})$

$\rho \frac{\partial \underline{v}}{\partial t} = -\nabla (W + \frac{v^2}{2}) + \underline{v} \times \underline{\omega}$
 \downarrow
 Magnus force (analogy $\nabla \times \underline{B}$)

$\frac{\partial \underline{\omega}}{\partial t} = \nabla \times (\underline{v} \times \underline{\omega})$

vorticity induction eqn.

$= -\underline{v} \cdot \nabla \underline{\omega} + \underline{\omega} \cdot \nabla \underline{v} - \underline{\omega} \nabla \cdot \underline{v}$

→ what is vorticity?

$$\underline{\omega} = \nabla \times \underline{v}$$

→ describes rotation of fluid element

→ $2 \otimes$ effective local angular velocity of fluid

i.e.
$$d\underline{v} = [\underline{\omega} \times \underline{r}] / 2$$

→ obvious analogy $\underline{B}, \underline{J} \leftrightarrow \underline{v}, \underline{\omega}$

- vorticity is critical as it controls the momentum evolution not governed by Bernoulli's law. This is generally the non-trivial part (i.e. roll-up, turbulence, etc.).

$$\frac{d\underline{\omega}}{dt} = \underline{\omega} \cdot \nabla \underline{v} - \underline{\omega} \nabla \cdot \underline{v}$$

Vorticity Evolution Equation
 $\nabla \cdot \underline{v} = 0$

if compressible: $\frac{d}{dt} \frac{\underline{\omega}}{\rho} = \frac{\underline{\omega} \cdot \nabla \underline{v}}{\rho}$

→ $\left\{ \begin{array}{l} \underline{\omega}/\rho \text{ frozen} \\ \text{into flow} \end{array} \right.$

Now, useful to define:

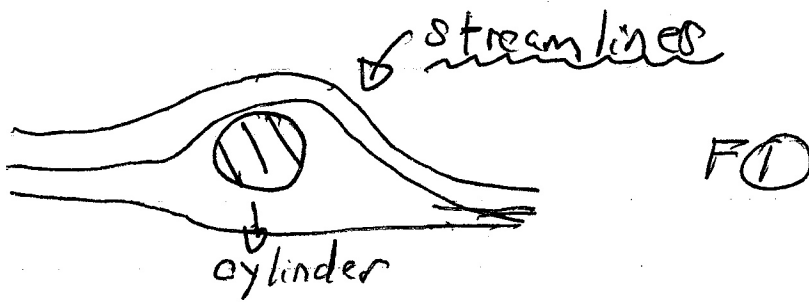
- stationary flow $\partial \underline{v} / \partial t = 0$

- streamline \equiv lines s/t tangent to streamline gives fluid velocity (i.e. \underline{v})

i.e. parametrize streamline via:

$$\frac{dx}{v_x} = \frac{dy}{v_y} = \frac{dz}{v_z}$$

eg.



For stationary flow:

$$0 = \underline{v} \times \underline{\omega} - \nabla \left(W + \frac{v^2}{2} \right)$$

frozen in \rightarrow vector

Consider line l moving with fluid particles. dl is segment

$$\rightarrow \underline{v} + dl \cdot \underline{\nabla} \underline{v}$$

$$\frac{d}{dt} \underline{h}_2 = \underline{v}(\underline{x} + dl, t)$$

$$\frac{d}{dt} \underline{h}_1 = \underline{v}(\underline{x}, t)$$

$$\underline{en} \ dt \quad \Delta dl = dt \ dl \cdot \underline{\nabla} \underline{v}$$

$$\underline{so} \ \frac{d}{dt} dl = dl \cdot \underline{\nabla} \underline{v} \quad \left. \vphantom{\frac{d}{dt} dl} \right\}$$

but \underline{w} obeys same eqn. as dl

\therefore w/p 'frozen into' fluid

Now, $\underline{l} \equiv$ unit vector tangent to streamline

$$0 = \underline{l} \cdot (\underline{v} \times \underline{\omega}) - \underline{l} \cdot \underline{\nabla} \left(W + \frac{v^2}{2} \right)$$

$$\underline{l} = \underline{v} / |\underline{v}|$$

$$\Rightarrow \frac{d}{ds} \left(W + \frac{v^2}{2} \right) = 0$$

\rightarrow statement of energy balance.

$$W + \frac{v^2}{2} = \text{const.}$$

static
Bernoulli's Eqn. y
const. diffn for each
streamline

Note if gravitational force,

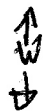
$$W + \frac{v^2}{2} + gz = \text{const.}$$

Application : Lift Theorem

Consider airplane wing
($l \gg w$)



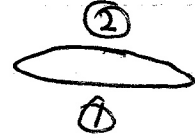
side



top



$$\frac{\rho}{\rho} + \frac{V^2}{2} = \text{const.}$$



⇒

$$\frac{\rho_1}{\rho} + \frac{V_1^2}{2} = \frac{\rho_2}{\rho} + \frac{V_2^2}{2}$$

(infinitesimally thin wing)

∴

$$\frac{\rho_1}{\rho} - \frac{\rho_2}{\rho} = \frac{V_2^2}{2} - \frac{V_1^2}{2}$$

$$= \frac{1}{2} (V_2 - V_1) (V_2 + V_1)$$

For infinitesimal wing:

$$\frac{V_1 + V_2}{2} \approx V_p \quad \text{speed of plane}$$

$$V_2 - V_1 = \frac{1}{W} \oint_{\text{wing}} \underline{v} \cdot d\underline{l}$$

defines circulation

$$\equiv \frac{\Gamma}{W} \quad (\text{circulation})$$

⇒

$$\frac{\rho_1 - \rho_2}{\rho} = \frac{\Gamma}{W}$$

$$F_{\text{Lift}} = (\rho_1 - \rho_2) A_{\text{wing}}$$

$$= \frac{\rho \Gamma}{W} (l w) V_p = \rho \Gamma l V_p$$

∴

$$F_{lift} = \rho \Gamma l V_p$$

lift theorem

$$\Gamma \equiv \oint_{wing} \underline{v} \cdot d\underline{s} \quad \leftrightarrow \text{circulation}$$

Note: - high air temp. \Rightarrow low ρ ∴ take-off difficult (Phoenix, summer '90)

- can empirically, theoretically relate

$$\Gamma = C_L V_p W$$

\hookrightarrow lift coefficient (\propto attack; shape; etc.)

$$\Rightarrow F_L = \rho A_{wing} C_L V_p^2$$

More generally:

Drag Force \equiv Force on object in direction of motion
 \hookrightarrow drag coefficient

Lift Force \equiv Force on object \perp direction of motion
 \hookrightarrow lift coefficient

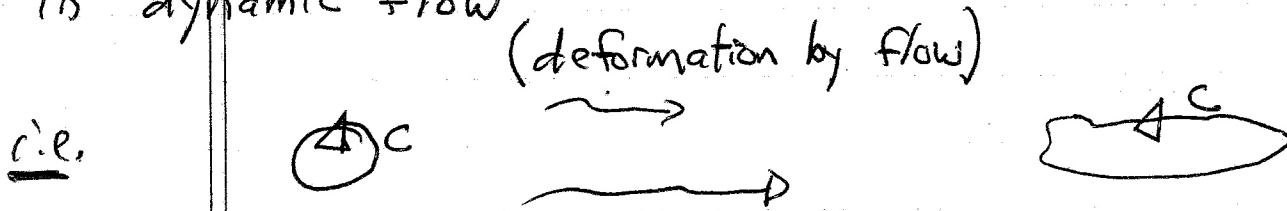
b.) Kelvin's Theorem

$$\Gamma = \oint_C \underline{v} \cdot d\underline{s} \quad \equiv \text{defines circulation about contour } C$$

obvious analogue \rightarrow
Kelvin's Theorem

15.

Consider evolution of circulation about contour
in dynamic flow



where:

- contour not broken (i.e. ideal fluid)
- C still encloses same fluid element

consider

Thus, need \sim

$$\frac{d}{dt} \oint_C \underline{v} \cdot d\underline{l} = d\Gamma/dt$$

where

- both contour and flow evolve

\Downarrow

- total derivative

For contour element $d\underline{l} = d(\underline{r}_2 - \underline{r}_1) \Big|_{\substack{\lim \\ \underline{r}_2 \rightarrow \underline{r}_1}} = d\underline{r}$

so

$$\frac{d}{dt} \oint \underline{v} \cdot d\underline{l} = \oint \frac{d\underline{v}}{dt} \cdot d\underline{l} + \oint \underline{v} \cdot \frac{d}{dt} d\underline{l}$$

$$= \oint d\underline{l} \cdot \frac{d\underline{v}}{dt} + \cancel{\oint \underline{v} \cdot d\underline{l}} \rightarrow \text{closed contour}$$

$$\frac{d\Gamma}{dt} = \oint \frac{d\underline{l}}{dt} \cdot \frac{d\underline{v}}{dt}$$

but for ideal fluid, $\frac{d\underline{v}}{dt} = -\underline{\nabla}W$

(ideal \Leftrightarrow isentropic)

$$\frac{d\Gamma}{dt} = -\oint d\underline{l} \cdot \underline{\nabla}W = 0$$

∞ Kelvin's Thm: In an ideal (isentropic) fluid, $\Gamma = \oint_C \underline{v} \cdot d\underline{l}$ (circulation) is conserved.

A.k.a. obvious analogy is Poincaré - Cartan invariant

$$I_p = \oint \underline{p} \cdot d\underline{q} \quad dI_p/dt = 0$$

useful also to note:

$$\Gamma = \oint_C \underline{v} \cdot d\underline{l} = \int_S d\underline{s} \cdot \underline{\omega}$$



$$2\pi r V_{rot} = \pi^2 \omega$$

$$V_{rot} = R \Omega_{rot} \quad \Omega_{rot} = \omega/2$$

$$\underline{\omega} = \underline{\nabla} \times \underline{v}$$

(vorticity)

S enclosed by C

∴ circulation = flux of vorticity thru surface enclosed by contour C.

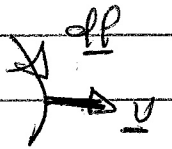
→ Magnetic Analogy, Helmholtz's Thm:

$$\frac{d}{dt} \int \underline{B} \cdot d\underline{q} = 0$$

$$\frac{-1}{c} \frac{\partial \underline{B}}{\partial t} = \underline{\nabla} \times \underline{E}, \quad \underline{E} + \frac{\underline{v} \times \underline{B}}{c} = \underline{j}$$

Now,

$$\frac{d}{dt} \int \underline{B} \cdot d\underline{q} = \int \frac{\partial \underline{B}}{\partial t} \cdot d\underline{q} + \int \underline{B} \cdot \frac{d\underline{q}}{dt}$$

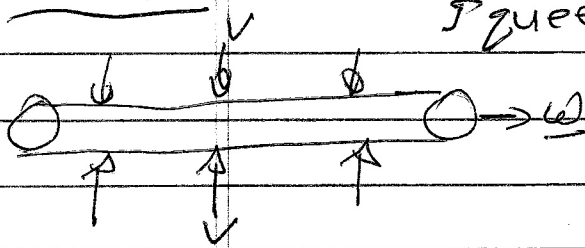


$$= -c \int \underline{\nabla} \times \underline{E} \cdot d\underline{q} + \int \underline{B} \cdot (\underline{v} \times d\underline{l})$$

$$= -c \int \underline{E} \cdot d\underline{l} - \int d\underline{l} \cdot (\underline{v} \times \underline{B})$$

$$= -c \int \underline{E} \cdot d\underline{l} + c \int \underline{E} \cdot d\underline{l} = 0 \quad \checkmark$$

Example:



Squeezed at $\underline{v} = v(y, t) \hat{y}$

compressed vortex tube
(ohv $\underline{\nabla} \cdot \underline{v} \neq 0$)

How does vorticity evolve?

$$\underline{\omega} \cdot \underline{\nabla} \underline{v} = 0$$

$$\frac{\omega}{\rho} A \sim \text{const.} \quad (\text{Kelvin})$$

$$\rho A \sim \text{const} \quad (\text{Mass})$$

\Rightarrow

$$\omega / \rho \sim \text{const.}$$

$$\therefore \omega \sim \rho \sim 1/A.$$

c.) Energy Equation and Energy Flux

For energy evolution, need consider:

$$\Sigma = \frac{\rho V^2}{2} + \rho E$$

\downarrow \downarrow
 kinetic internal
 energy energy density
 density (i.e. thermal)

As with momentum density, consider $\partial \Sigma / \partial t$

① ②

$$\frac{\partial \Sigma}{\partial t} = \frac{\partial}{\partial t} \left(\frac{\rho V^2}{2} + \rho E \right)$$

①:

$$\frac{\partial}{\partial t} \left(\frac{\rho V^2}{2} \right) = \frac{V^2}{2} \frac{\partial \rho}{\partial t} + \rho \underline{V} \cdot \frac{\partial \underline{V}}{\partial t}$$

$$= \underbrace{-\frac{1}{2} V^2 \underline{\nabla} \cdot (\rho \underline{V})}_{\substack{\downarrow \\ \text{from eqn.} \\ \text{Continuity}}} - \underbrace{\underline{V} \cdot \underline{\nabla} \rho + \rho \underline{V} \cdot (\underline{V} \cdot \underline{\nabla} \underline{V})}_{\substack{\downarrow \\ \text{from momentum balance}}}$$

now

$$\underline{V} \cdot \underline{\nabla} \underline{V} = -\underline{V} \times \underline{\omega} + \frac{1}{2} \underline{\nabla} V^2$$

⇒

$$\rho \underline{v} \cdot \underline{v} \cdot \underline{\nabla} \underline{v} = \rho \underline{v} \cdot \left(-\underline{v} \times \underline{\omega} + \frac{1}{2} \underline{\nabla} v^2 \right)$$

$$= \rho \underline{v} \cdot \underline{\nabla} \frac{v^2}{2}$$

Also $dW = T ds + v dp$

$$= T ds' + \frac{dv^2}{\rho}$$

$$\Rightarrow \underline{\nabla} p = \rho \underline{\nabla} W - \rho T \underline{\nabla} s'$$

$$\therefore \frac{\partial}{\partial t} \left(\frac{\rho v^2}{2} \right) = -\frac{v^2}{2} \underline{\nabla} \cdot (\rho \underline{v}) - \rho \underline{v} \cdot \underline{\nabla} \left(\frac{v^2}{2} + W \right)$$

$$+ \rho T \underline{v} \cdot \underline{\nabla} s'$$

② $\frac{\partial}{\partial t} (\rho E)$

Useful to transform using thermodynamic identity

$$dE = dQ - p dv$$

$$= T ds - p dv$$

$$\text{out } v = 1/\rho$$

$$dv = -\frac{d\rho}{\rho^2}$$

⇒

$$dE = Tds + \frac{p}{\rho^2} d\rho$$

$$\text{so } d(\rho E) = \rho dE + E d\rho$$

$$= \rho \left(Tds + \frac{p}{\rho^2} d\rho \right) + E d\rho$$

$$= \left(\frac{p}{\rho} + E \right) d\rho + \rho Tds$$

$$\text{but } E + \frac{p}{\rho} = E + \rho v = W$$

$$\therefore d(\rho E) = W d\rho + \rho Tds$$

$$\Rightarrow \frac{\partial}{\partial t} (\rho E) = W \frac{\partial \rho}{\partial t} + \rho T \frac{\partial s}{\partial t}$$

$$= -W \nabla \cdot (\rho \underline{v}) - \rho T \nabla \cdot \underline{v} S$$

↓
from continuity

↓
from a diabatic equation

Thus, combining ①, ②

$$\begin{aligned}
 \frac{\partial}{\partial t} \left(\frac{\rho v^2}{2} + \rho \phi \right) &= - \frac{v^2}{2} \nabla \cdot (\rho \underline{v}) - \rho \underline{v} \cdot \nabla \left(\frac{v^2}{2} + w \right) \\
 &\quad + \cancel{\rho T \underline{v} \cdot \nabla S} - \cancel{w \nabla \cdot (\rho \underline{v})} - \cancel{\rho T \underline{v} \cdot \nabla S} \\
 &= - \left(\frac{v^2}{2} + w \right) \nabla \cdot (\rho \underline{v}) - \rho \underline{v} \cdot \nabla \left(\frac{v^2}{2} + w \right) \\
 &= - \nabla \cdot \left(\rho \underline{v} \left(\frac{v^2}{2} + w \right) \right) \\
 &= - \nabla \cdot \left(\underline{v} \left(\frac{\rho v^2}{2} + w \right) \right)
 \end{aligned}$$

Thus, have:

$$\frac{\partial}{\partial t} \left(\frac{\rho v^2}{2} + \rho \phi \right) + \nabla \cdot \left(\rho \underline{v} \left(\frac{v^2}{2} + w \right) \right) = 0$$

\Rightarrow

$$\frac{\partial}{\partial t} \int dV \left(\frac{\rho v^2}{2} + \rho \phi \right) = - \int dV \nabla \cdot \left(\rho \underline{v} \left(\frac{v^2}{2} + w \right) \right)$$

Change in energy
in volume V

$$= - \int d\underline{S} \cdot \left[\rho \underline{v} \left(\frac{v^2}{2} + w \right) \right]$$

energy flux thru \underline{S} , enclosing V

$$\text{Energy density flux} \equiv \rho \underline{v} \left(\frac{v^2}{2} + W \right)$$

(thermal)

Interesting to note:

→ unit mass of fluid carries energy $\frac{v^2}{2} + W$ (not

$\frac{v^2}{2} + E$) in motion

→ from

$$W = E + PV$$

$$= E + \frac{P}{\rho}$$

Then, for energy flux thru surface:

$$\int d\underline{S} \cdot \rho \underline{v} \left(\frac{v^2}{2} + W \right) = \int d\underline{S} \cdot \rho \underline{v} \left(\frac{v^2}{2} + E + \frac{P}{\rho} \right)$$

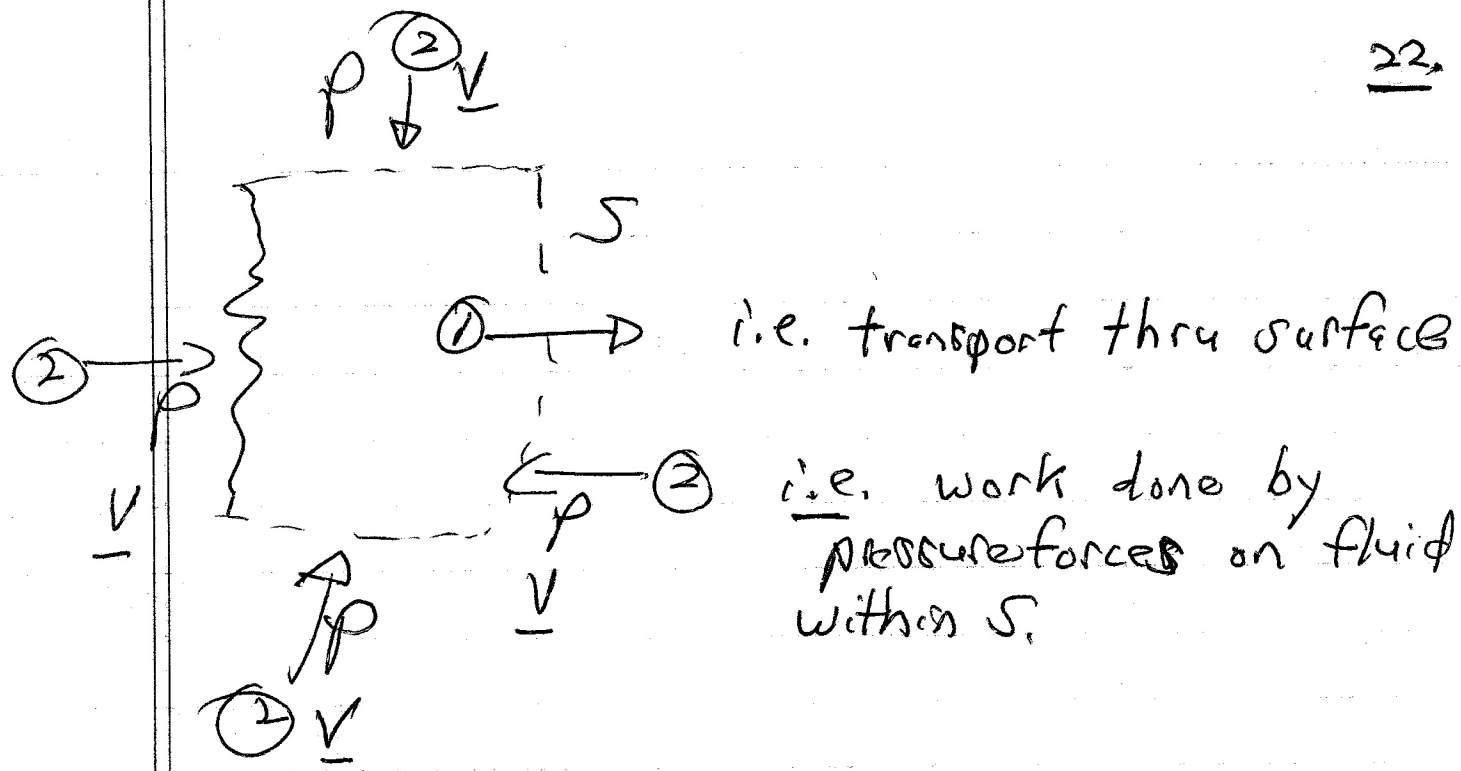
①

$$= \int d\underline{S} \cdot \rho \underline{v} \left(\frac{v^2}{2} + E \right)$$

↑
↳ naive expectation
② (kinetic + internal energy transported)

$$+ \int d\underline{S} \cdot \underline{v} P$$

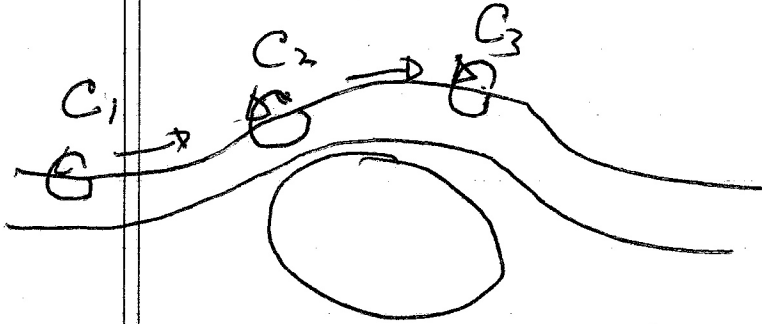
↓
work done by pressure forces
on fluid within surface.



~~Control Volume Flow in Ideal Incompressible Fluid~~

(c) Potential Flow

- Consider fluid streamlines:



if $\underline{\omega} = 0$ at any point along streamline, then Kelvin's thm $\Rightarrow \underline{\omega} = 0$ everywhere on streamline.

Easily seen by considering circulation around infinitesimal loop "pulled" along streamline. Thus, if

$$\oint_{C_1} \underline{v} \cdot d\underline{l} = \int_{A_1} \underline{\omega} \cdot d\underline{s} = 0, \text{ then } \oint_{C_n} \underline{v} \cdot d\underline{l} = \int_{A_n} \underline{\omega} \cdot d\underline{s} = 0$$

for all C_n .

- flow with $\underline{\omega} = \nabla \times \underline{v} = 0$ in all space is defined as:

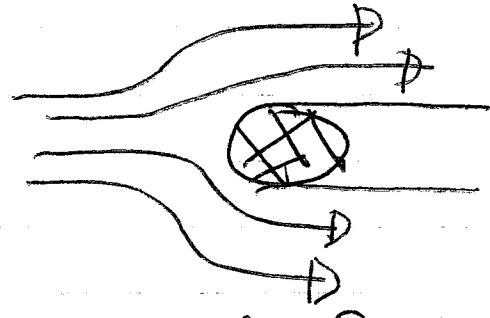
\Rightarrow potential, irrotational flow

\Leftarrow $\underline{\omega} \neq 0$ rotational, vortical flow

- Important to note breakdown of Kelvin's Thm

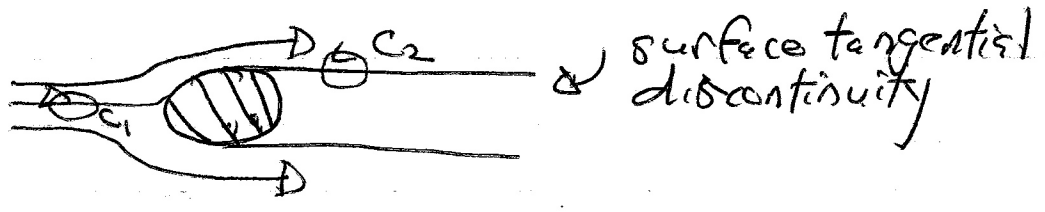
applicability, namely to flows with separation

i.e. consider flow around sphere



- i.e.
- streamlines separate from body
 - surface of tangential discontinuity appears (velocity component)
- ⇒
- Kelvin Thm not applicable

i.e.



- cannot infer $\oint_a \underline{v} \cdot d\underline{l}$ from $\oint_c \underline{v} \cdot d\underline{l}$ due to separation-induced tangential discontinuity.
- Also, viscosity important in (boundary layer) region of discontinuity. As viscous effects $\sim \mu k^2$, deviation from potential flow naturally most significant in small scales region of boundary layer!

Now, for isentropic fluids:

$$\partial \underline{v} / \partial t + \underline{v} \cdot \nabla \underline{v} = -\nabla W$$

$W \equiv$ enthalpy

stream function

for potential flow, $\underline{v} = \underline{\nabla}\phi \Rightarrow \underline{\omega} = 0$

$$\underline{v} \cdot \underline{\nabla} \underline{v} = -\underline{v} \times \underline{\nabla} \phi + \underline{\nabla} (v^2/2)$$

$$= 0 + \underline{\nabla} (v^2/2), \text{ for potential flow}$$

$$\frac{\partial \underline{v}}{\partial t} + \underline{\nabla} (v^2/2) = -\underline{\nabla} W$$

$$\underline{v} = \underline{\nabla}\phi$$

$$\Rightarrow \underline{\nabla} \left(\frac{\partial \phi}{\partial t} + \frac{(\underline{\nabla}\phi)^2}{2} + W \right) = 0$$

have equation for dynamics of potential flow:

$$\frac{\partial \phi}{\partial t} + \frac{(\underline{\nabla}\phi)^2}{2} + W = f(t)$$

f(t) defined for each streamline

- for $\partial\phi/\partial t = 0$, recover Bernoulli's Law

- obvious that potential not uniquely defined, as $\underline{v} = \underline{\nabla}\phi$

Now, consider incompressible fluid potential flow,
i.e.

- flows leaving density constant (no compression, expansion)

$$- \underline{\nabla} \cdot \underline{v} = 0 \quad \Leftrightarrow \quad \frac{d\rho}{dt} = 0 \quad (\rho \text{ constant})$$

$$\Rightarrow \text{if } \underline{v} = \underline{\nabla} \phi$$

$$\Rightarrow \nabla^2 \phi = 0$$

$$\frac{\partial \phi}{\partial t} + \frac{(\underline{\nabla} \phi)^2}{2} + \frac{p}{\rho} = f(t)$$

\therefore for static flow, with gravity, \Rightarrow Bernoulli Eqn.:

$$v^2/2 + p/\rho + gz = \text{const.}$$

Criterion for "incompressibility":

- "incompressibility" is valid description for certain classes of flows, dependent on time scales, speeds, etc.

- for stationary flows

$$\partial \underline{v} / \partial t = 0,$$

$$\frac{p}{\rho} + \frac{v^2}{2} = \text{const.}$$

Now, for {adiabatic} fluid (T const) \Rightarrow
 {isentropic}

$$\Delta P = \left(\frac{\partial p}{\partial \rho} \right)_s \Delta \rho$$

but $\rho + \frac{v^2}{2} = \text{const.}$

$$\Rightarrow \Delta \left(\frac{v^2}{2} \right) = - \left(\frac{\partial p}{\partial \rho} \right)_s \frac{\Delta \rho}{\rho}$$

"Incompressibility" $\Rightarrow \Delta P / \rho \ll 1$

$$\left(\frac{\partial p}{\partial \rho} \right)_s = c_s^2 \quad (\text{sound speed in fluid})$$

$$\therefore v^2 / c_s^2 \ll 1 \Rightarrow \text{Flow } \underline{\text{incompressible}}.$$

Note: {Supersonic flows always compressible \Rightarrow
 Fluid dynamics coupled to acoustic waves.

- for dynamic flows (more generally);

need compare terms in continuity equation

$$\frac{d\rho}{dt} = -\rho \nabla \cdot \underline{v}$$

Now $\tau \rightarrow$ time scale for flow
 $l \rightarrow$ spatial scale for flow

Then, $\frac{dp}{dt} \sim \frac{\Delta p}{\tau}$

$\rho \nabla \cdot \underline{v} \sim \rho \frac{\tilde{v}}{l}$

To relate Δp to \tilde{v} , consider Euler equation:

$\frac{\partial \underline{v}}{\partial t} = - \frac{\nabla p}{\rho}$

$\Rightarrow \frac{\tilde{v}}{\tau} \sim \frac{c_s^2 \Delta p}{\rho l}$ (tacitly, $\tilde{v} \approx l/\tau$)

$\tilde{v} \sim \tau \frac{c_s^2 \Delta p}{\rho l}$

$\therefore \frac{dp}{dt} \sim \left(\frac{c_s^2 \rho}{l} \right)^{-1} \frac{\tilde{v}}{\tau}$

$\rho \nabla \cdot \underline{v} \sim \frac{\tilde{v}}{l} \rho$

$\nabla \cdot \underline{v} \approx 0$ if $\frac{dp}{dt} \ll \rho \nabla \cdot \underline{v}$

$$\frac{\tilde{v}}{l} \gg \frac{l \rho}{c_s^2 \gamma^2} \tilde{v} \Rightarrow c_s^2 \gg \frac{l^2}{\gamma^2}$$

Thus, dynamics is compressible if $\begin{cases} c_s^2 \gg l^2/\gamma^2 \\ \gg u^2/k^2 \text{ (wave)} \end{cases}$

- note can synthesise static, dynamic conditions to obtain incompressibility criterion:

$$c_s^2 > \begin{cases} \tilde{v}^2 \\ l^2/\gamma^2 \end{cases} \quad \text{i.e. } \begin{cases} \text{time slow compared to} \\ \text{time to traverse 1 spatial} \\ \text{scale at acoustic speeds.} \end{cases}$$

Some further facts about potential flows (generally incompressible):

- for body (i.e. rigid sphere) immersed in fluid, if amplitude oscillation \ll dimensions of body \Rightarrow motion describable by potential flow

i.e. $a \equiv$ amplitude motion

$u \equiv$ body velocity

$f \equiv$ frequency of oscillation

$l \equiv$ size of body

Simply compare $\frac{\partial \mathbf{v}}{\partial t}$ to $\mathbf{v} \cdot \nabla \mathbf{v}$, noting

$$\text{if } \frac{\partial \underline{v}}{\partial t} \gg \underline{v} \cdot \nabla \underline{v} \Rightarrow \frac{\partial \underline{v}}{\partial t} \approx -\nabla w$$

$$\text{so } \nabla \times \underline{v} = 0 \Rightarrow \left\{ \begin{array}{l} \text{Potential} \\ \text{flow} \end{array} \right.$$

$$\text{Now } \omega \sim u/a$$

$$\frac{\partial \underline{v}}{\partial t} = -i\omega \underline{v} \sim u^2/a \quad (\underline{v} \sim u \text{ near body})$$

$$\underline{v} \cdot \nabla \underline{v} \sim u^2/l \quad (l \text{ sets smallest scale in problem})$$

$$\left| \frac{\partial \underline{v}}{\partial t} \right| \gg \left| \underline{v} \cdot \nabla \underline{v} \right| \Rightarrow \frac{u^2}{a} \gg \frac{u^2}{l}$$

$$\Rightarrow l \gg a$$

Thus, fluid dynamics resulting from small oscillation of body describable by potential flow.

- In potential flow, streamlines must be open, not closed.

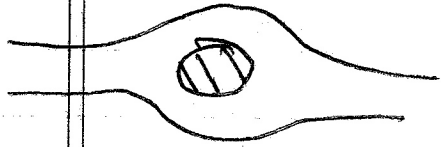
To see, consider circulation about closed contour

$$\oint_C \underline{v} \cdot d\underline{l} = \int_{S_{\text{enc}}} \underline{\omega} \cdot \underline{n} = 0$$

$\underline{\omega} = 0$ for potential flow

but, by definition, $\int_{\text{streamline}} \underline{v} \cdot d\underline{l} \neq 0 \Rightarrow$ streamlines must be open!

c.e.



sphere in $\underline{v} = v_0 \hat{z}$ flow is typical potential flow problem (describes flow at distance from sphere).

- For incompressible flow, (not potential)

$$\frac{d\underline{\omega}}{dt} = \underline{\omega} \cdot \nabla \underline{v} - \underline{\omega} \nabla \cdot \underline{v}$$

In $\underline{2D}$, $\underline{\omega} \cdot \nabla \underline{v} = 0$ c.e. $\left\{ \begin{array}{l} \underline{v} = (v_x(x,y), v_y(x,y)) \\ \underline{\omega} = \omega_z(x,y) \hat{z} \end{array} \right.$

Then, $\frac{d\underline{\omega}}{dt} = 0$

Now, $\nabla \cdot \underline{v} = 0 \Rightarrow \begin{array}{l} v_x = \partial \psi / \partial y \\ v_y = -\partial \psi / \partial x \end{array}$

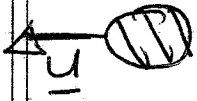
$$\underline{\omega} = \left(\frac{\partial v_y}{\partial x} - \frac{\partial v_x}{\partial y} \right) \underline{\hat{z}} = \underline{\hat{z}} (-\nabla^2 \psi)$$

$$\frac{d\underline{\omega}}{dt} = 0 \Rightarrow \begin{cases} + \frac{\partial}{\partial t} \nabla^2 \psi + \underline{\nabla} \psi \times \underline{z} \cdot \nabla \nabla^2 \psi = 0 \\ 2D \text{ incompressible fluid eqn.} \end{cases}$$

iv.) Problems in Potential Flow

a.) Incompressible Potential Flow Around Sphere

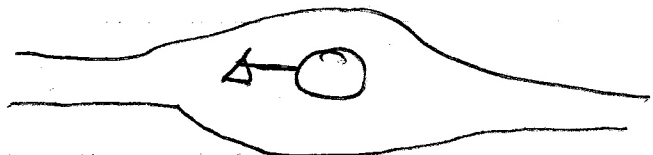
Consider ^{rigid} sphere in motion at \underline{u} in infinite fluid



Flow Pattern ?

Now :

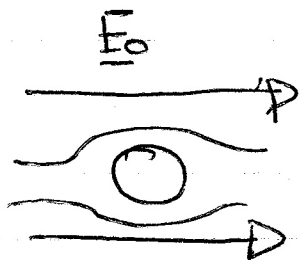
- intuitively, expect :



i.e. equivalent to $\begin{cases} \text{sphere at rest} \\ \underline{v}|_{\text{fluid}} = -\underline{u} \\ \infty \end{cases}$

Electrostatic analogy: Conducting sphere in uniform electric field

i.e.



$$\phi = -\underline{E}_0 \cdot \underline{r} + \phi_{\text{sphere}}$$

ϕ_{sphere} is dipole field.

Dipole moment determined by b.c.

i.e. $\phi = \text{const} = 0$ on sphere surface

Now, for potential flow (incompressible):

$$\nabla^2 \phi = 0$$

$$\underline{v} = \underline{\nabla} \phi$$

$$v_n = \underline{v} \cdot \hat{n} = \underline{u} \cdot \hat{n} \Big|_{\text{surface}}$$

(i.e. normal velocity = sphere velocity on surface)

By analogy with electrostatics, can solve via:

- multipole expansion
- b.c.'s determine effective "charge" distribution

Recall e.s. $\Rightarrow \nabla^2 \phi = -4\pi\rho$

$$\phi = \int d^3x' \frac{\rho(x')}{|x-x'|}$$

For \underline{x} outside region ρ :

$$\phi(\underline{x}) = \int d^3x' \frac{\rho(x')}{|x-x'|}$$

$$= \int d^3x' \frac{\rho(x')}{|x-x'|} = \int d^3x' \rho(x') \cdot \nabla \left(\frac{1}{|x|} \right) + \dots$$

$$= \underbrace{\frac{Q}{|x|}}_{\text{monopole}} - \underbrace{d \cdot \nabla \left(\frac{1}{|x|} \right)}_{\text{dipole}} + \dots \underbrace{\dots}_{\text{quadrupole}}$$

Thus, can write down general solution for potential flow streamlines around body as multipole expansion.

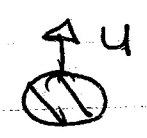
$Q = 0$ (no sources, sinks)

in general dipole dominates

in 2D, same story with $\ln|x-x'| \rightarrow 1/|x-x'|$

Here: $\underline{u} = u \hat{z}$ (spherical symmetry) (flow velocity) (body velocity)

$V_n|_R = V_r|_R = u \hat{z} \cdot \hat{n} = u \cos\theta$ } boundary condition



Now, $\phi(\underline{x}) = \underline{A} \cdot \underline{\nabla} (1/|\underline{x}|)$

$\underline{A} = A \hat{z}$ (dipole moment in \hat{z} direction)

$\phi = -A \frac{\cos\theta}{r^2}$

$V_r = 2A \cos\theta / r^3$

$\Rightarrow \frac{2A \cos\theta}{R^3} = u \cos\theta$

$\Rightarrow A = \frac{R^3}{2} u$

$\phi = -u R^3 \cos\theta / 2 r^2$

$\underline{V} = \underline{\nabla} \phi$

determined general flow field

Note:

- can recover from $\phi = \sum \left(\frac{a}{r^l} + \frac{b}{r^{l+1}} \right) P_l(\cos\theta)$ ^{regularity at ∞}
 expansion and b.c.'s.

- if sphere in uniform field:

$$\phi = U_0 r \cos\theta + \phi_{\text{sphere}}$$

\downarrow
 determine from $V_n = 0$

- to determine pressure distribution on sphere,

Recall: $\rho \frac{\partial \phi}{\partial t} + \frac{\rho v^2}{2} + p = p_0$ } incompressible
 Bernoulli Eqn.
 \downarrow
 ambient pressure at ∞

Thus, can immediately write:

$$p(x) = p_0 - \frac{\rho}{2} \nabla \phi \cdot \nabla \phi - \rho \frac{\partial \phi}{\partial t}$$

$\phi(x) \equiv$ determined a/c' above via $\nabla^2 \phi = 0$
 and b.c.'s.

As sphere in motion (but uniform):

$$\frac{\partial \phi}{\partial t} = -\underline{u} \cdot \nabla \phi + \frac{\partial \phi}{\partial \underline{u}} \cdot \dot{\underline{u}} \quad \Delta \underline{u} = 0$$

so

$$p(\underline{x}) = p_0 - \frac{\rho}{2} \nabla \phi \cdot \nabla \phi - \underline{u} \cdot \nabla \phi$$

Generally, leads to concept of stagnation point

i.e. for Bernoulli Egn. for incompressible fluid:

$$\frac{p}{\rho} + \frac{v^2}{2} = \text{const.} = p_0$$

Now, consider fixed body in fluid with $\begin{cases} v_{\infty} = u_0 \\ p_{\infty} = p_0 \end{cases}$

As $v = 0$ on surface body:

$$p_{\text{max}} = p|_{\text{bdy}} = p_0 + \frac{1}{2} \rho u^2$$

- stagnation point ($v=0$) on body is point of maximal pressure

- maximal pressure determined by $\begin{cases} p_0 \\ \text{speed} \end{cases}$



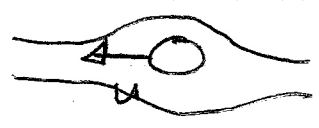
→ Fish skeleton strongest on front face, weakest elsewhere

↔ front face is point of maximal pressure ('head')

↔ eye lens adjusts to allow for speed-induced pressure changes.

b.) Drag Force and Induced Mass

→ Heuristics: Consider rigid body in water.



Slow body motion ⇒ potential flow around sphere
⇒ energy in fluid motion, too!

Thus, for F_{ext} to move body in fluid, need work against
- inertia of body (obvious)
- inertia of fluid, excited into potential flow

Thus, for body in water, need interpret Newton's 2nd Law as:

$$\underline{F}_{ext} = M_{eff} \frac{d\underline{y}}{dt}$$

$$M_{eff} = \underbrace{M}_{\substack{\downarrow \\ \text{mass of body}}} + m_{induced}$$

\Rightarrow induced mass of fluid in potential flow around body
 (mass of fluid flow which 'addresses' the body)

To calculate induced mass:

⊕ - calculate energy in potential flow around rigid body in uniform motion in fluid

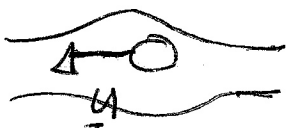
⊗ - use $dE = d\underline{p} \cdot \underline{y}$ to determine momentum in fluid

as $\underline{p} = \underline{p}(\underline{y}) \Rightarrow p_i = m_{ik} U_k$

$\therefore m_{ik}$ is induced mass tensor!

\rightarrow Calculation: Consider rigid body moving in fluid

c.e.



Now, for flow field outside body, multipole expansion solution to $\nabla^2 \phi = 0$ yields

$$\phi = \frac{\phi}{r} + \underline{A} \cdot \underline{\nabla} \left(\frac{1}{r} \right) + \dots$$

$\frac{\phi}{r}$ monopole (vanishes \rightarrow no sources)
 $\underline{A} \cdot \underline{\nabla} \left(\frac{1}{r} \right)$ dipole (dominant multipole at large radius)

\rightarrow dipole moment: $A = c R^3 \underline{u}$

$\therefore \phi = \underline{A} \cdot \underline{\nabla} \left(\frac{1}{r} \right)$ ($c = 1/2$, sphere)

$$= - \underline{A} \cdot \underline{r} / r^3 = - \underline{A} \cdot \hat{n} / r^2$$

$$\underline{v} = \underline{\nabla} \phi = \underline{A} \cdot \underline{\nabla} \nabla \left(\frac{1}{r} \right)$$

$$= (\underline{A} \cdot \underline{\nabla}) \left(- \underline{r} / r^3 \right)$$

$$\underline{v} = (3(\underline{A} \cdot \hat{n}) \hat{n} - \underline{A}) / r^3$$

Now, for energy, seek calculate fluid energy in volume V enclosed within radius R around body. Take $R^3 \gg V_0 \equiv$ volume of body.

Thus:

$$E = \frac{1}{2} \rho \int dV | \underline{v} |^2$$

$$= \frac{1}{2} \rho \int d^3x \left(\underline{u}^2 + |\hat{n}|^2 - \underline{u}^2 \right)$$

$$\begin{aligned} \text{out } \nabla^2 u^2 &= (\underline{v} + \underline{u}) \cdot (\underline{v} - \underline{u}) \\ &= \nabla(\phi + \underline{u} \cdot \underline{r}) \cdot (\underline{v} - \underline{u}) \\ &= \nabla \cdot [(\phi + \underline{u} \cdot \underline{r})(\underline{v} - \underline{u})] \end{aligned}$$

as $\underline{v} = \nabla\phi$ $\nabla \cdot \underline{v} = 0$
 $\underline{u} = \text{const.}$ $\nabla \cdot \underline{u} = 0$

$$\therefore E = \frac{1}{2} \rho \int d^3x \left[u^2 + \nabla \cdot [(\phi + \underline{u} \cdot \underline{r})(\underline{v} - \underline{u})] \right]$$

$$= \frac{1}{2} \rho u^2 (V - V_0) + \frac{1}{2} \rho \int d\underline{s} \cdot [(\phi + \underline{u} \cdot \underline{r})(\underline{v} - \underline{u})]$$

Volume space
Volume object/body

$$V = \frac{4\pi}{3} R^3$$

$$\begin{cases} (\underline{v} - \underline{u}) \cdot d\underline{s} = 0 \\ \text{on } R_0 \text{ surface} \end{cases}$$

Now, $d\underline{s} = \hat{n} R^2 d\Omega$, on outer surface

$$E = \frac{1}{2} \rho u^2 (V - V_0)$$

$$+ \frac{1}{2} \rho \int R^2 d\Omega [(\hat{n} \cdot \underline{v} - \hat{n} \cdot \underline{u})(\phi + \underline{u} \cdot \underline{r})]$$

$$E = \frac{1}{2} \rho u^2 (V - V_0) + \frac{1}{2} \rho \int R^2 d\Omega \left[\left(2 \frac{(\underline{A} \cdot \underline{\hat{n}})}{R^3} - \underline{u} \cdot \underline{\hat{n}} \right) \left(-\frac{\underline{A} \cdot \underline{\hat{n}}}{R^2} + R \underline{u} \cdot \underline{\hat{n}} \right) \right]$$

$$= \frac{1}{2} \rho u^2 (V - V_0) + \frac{1}{2} \rho \int R^2 d\Omega \left[-2 \frac{(\underline{A} \cdot \underline{\hat{n}})^2}{R^5} \right] \quad \begin{array}{l} \text{vanishes} \\ \text{for large } R \end{array}$$

$$+ \frac{(\underline{u} \cdot \underline{\hat{n}})(\underline{A} \cdot \underline{\hat{n}})}{R^2} + \frac{2(\underline{A} \cdot \underline{\hat{n}})(\underline{u} \cdot \underline{\hat{n}}) - R(\underline{u} \cdot \underline{\hat{n}})^2}{R^2}$$

$$= \frac{1}{2} \rho u^2 (V - V_0) + \frac{1}{2} \rho \int R^2 d\Omega \left[\frac{3(\underline{A} \cdot \underline{\hat{n}})(\underline{u} \cdot \underline{\hat{n}})}{R^2} - R(\underline{u} \cdot \underline{\hat{n}})^2 \right]$$

Thus finally,

$$E = \frac{1}{2} \rho u^2 (V - V_0) + \frac{1}{2} \rho \int d\Omega \left[3(\underline{A} \cdot \underline{\hat{n}})(\underline{u} \cdot \underline{\hat{n}}) - R^3 (\underline{u} \cdot \underline{\hat{n}})^2 \right]$$

$$d\Omega = d\theta \sin\theta d\phi$$

$$\int d\Omega () = \langle () \rangle$$

$$\Rightarrow \langle (\underline{A} \cdot \underline{\hat{n}})(\underline{B} \cdot \underline{\hat{n}}) \rangle = \frac{1}{2} \delta_{ij} A_i B_j = \frac{1}{3} \underline{A} \cdot \underline{B}$$

$$E = \frac{1}{2} \rho u^2 (V - V_0) + \frac{1}{2} \rho \left[4\pi A \cdot \underline{y} - \frac{4\pi}{3} R^3 u^2 \right]$$

$$= \frac{1}{2} \rho \left[4\pi A \cdot \underline{y} - u^2 V_0 \right]$$

Thus finally,

$$E = \frac{\rho}{2} \left[4\pi A \cdot \underline{y} - u^2 V_0 \right]$$

energy in
potential
flow and
body

Now, $\underline{A} = \underline{A}(u) \Rightarrow \left\{ \begin{array}{l} E = \frac{1}{2} m_{ik} u_i u_k \\ \text{defines induced mass} \\ \text{tensor} \end{array} \right.$

$$dE = \underline{y} \cdot d\underline{P}$$

$$\Rightarrow \underline{P} = \frac{\rho}{4\pi} \left[4\pi \underline{A} - V_0 \underline{y} \right]$$

momentum in
potential flow

Now, consider external force acting system, where system = body + fluid (in Pot. flow)

$$\text{i.e. } \underline{F}_{\text{ext}} = \frac{d\underline{P}_{\text{fluid}}}{dt} + M_{\text{body}} \frac{d\underline{U}}{dt}$$

$$\Rightarrow \underline{F}_e = (M_{dik} + m_{ik}) \frac{dU_k}{dt}$$

\therefore effective mass of "system" is sum of - body mass

- induced mass of fluid in potential flow around body

\rightarrow Note induced mass is determined purely by body shape (i.e. via volume and dipole moment)

$$\text{i.e. for sphere } \underline{A} = \frac{R_0^3}{2} \underline{U}$$

$$\underline{P} = \rho \left[4\pi \frac{R_0^3}{2} \underline{U} - \frac{4\pi}{3} R_0^3 \underline{U} \right]$$

$$= \rho \frac{2}{3} \pi R_0^3 \underline{U}$$

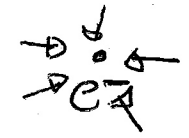
$$m_{\text{induced}} = \rho \frac{2}{3} \pi R_0^3$$

In general $M_{induced} \sim \# \rho R^3$

$\sim \# \rho V_0$
↓ \hookrightarrow displaced mass
numerical fluid
factor, shape dependent

→ Example of "renormalization" in classical physics
"dressing field" in continuum i.e. {renorm. polarization, debye shield, etc}

c.e. in quantum electrodynamics → electron polarizes vacuum



$$\rightarrow m_e = m_e^{bare} + m_e^{V.P.}$$

($E=mc^2$)

in classical potential flow → moving a sphere in H_2O requires that some energy go into surrounding media (the water!)

(skip)

→ Enhanced inertia due induced mass may alternatively, be viewed as drag force on body
mom. transmittal to fluid (careful of phase!)

c.e. $F_{ext} = \frac{dP_{fluid}}{dt} + M \frac{dv}{dt}$

$$M \frac{dy}{dt} = \underline{f_{ext}} - \frac{dP_{fluid}}{dt}$$

drag!

$$= \underline{f_{ext}} + \underline{f_{drag, lift}}$$

$f_{drag} \sim u$

$f_{drag} = -\frac{dP_{fluid}}{dt}$, along direction motion.

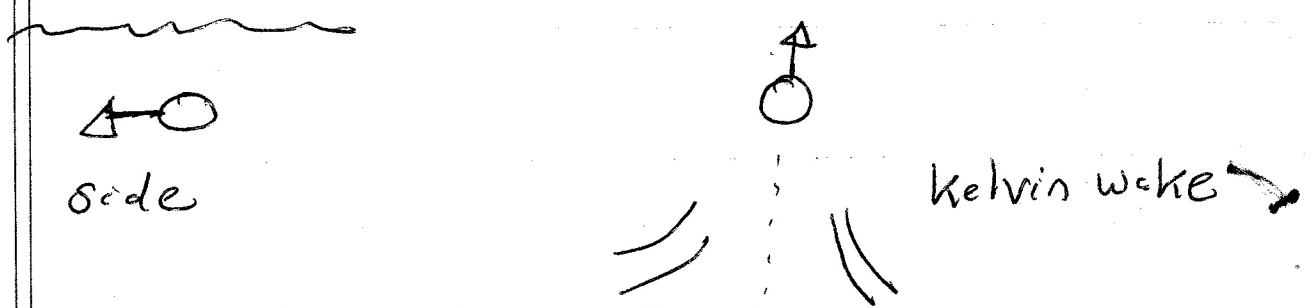
$f_{lift} = -\frac{dP_{fluid}}{dt}$, \perp direction of motion.

Note: \rightarrow if body is uniform motion in ideal (fantasy) fluid $f_{drag} = f_{lift} = 0$ } D'Alembert's Paradox

\rightarrow need external force to maintain uniform motion

as no = no dissipation (ideal fluid)
 = no loss of energy to ∞ ($V \sim 1/R^3$)

\rightarrow but if body near surface



body will radiate surface waves to ∞ (wake) \Rightarrow wave drag induced energy loss!

Related Problem:

- consider body in fluid, which is set in motion by external agent



Relate \underline{u} body to \underline{v} fluid!?

- Now $\underline{v} \equiv$ velocity of unperturbed flow

$$\frac{\|\nabla \underline{v}\| R_0}{\|\underline{v}\|} \ll 1 \Rightarrow \underline{v} \sim \text{const over scale of body} \\ (\text{potential flow valid})$$

so if body fully carried along by fluid ($\underline{v} = \underline{u}$), then force on it would equal force on volume of displaced fluid

$$\text{i.e. } \frac{d}{dt} (M \underline{u}) = \rho V_0 \frac{d \underline{v}}{dt}$$

but body moves relative to fluid, so that fluid acquires momentum \rightarrow drag due relative motion

$$\text{i.e. } d\underline{p}_{\text{fluid}}/dt = -\underline{m} \cdot \frac{d}{dt} [\underline{u} - \underline{v}]$$

∴ so really,

$$\frac{d}{dt} (M \underline{u}) = \rho \cdot V_0 \frac{d\underline{v}}{dt} - \underline{m} \cdot \frac{d}{dt} (\underline{u} - \underline{v})$$

$$\frac{d}{dt} (M u_i) = \rho V_0 \frac{d v_i}{dt} - m_{ik} \frac{d}{dt} (u_k - v_k)$$

⇒

$$M u_i = \rho V_0 v_i - m_{ik} (u_k - v_k)$$

$$(M \delta_{ik} + m_{ik}) u_k = (\rho V_0 \delta_{ik} + m_{ik}) v_k$$

$$u_k = \left(\frac{\rho V_0 \delta_{ik} + m_{ik}}{M \delta_{ik} + m_{ik}} \right) v_k$$

Note: $\rho V_0 < M$ (body heavier than displaced fluid) → body lags

$\rho V_0 > M$ → body leads

$\rho V_0 = M$ $u_k = v_k$.

Thus

$$M \frac{du}{dt} = \rho_f V \frac{dv}{dt} - \underline{m} \cdot \frac{d}{dt} [u - v]$$

$$(M \delta_{ij} + m_{ij}) \frac{du_j}{dt} = M_F \delta_{ij} + m_{ij} \frac{dv_j}{dt}$$

$$\therefore u_j = \left[(M_F \delta_{ij} + m_{ij}) / (M \delta_{ij} + m_{ij}) \right] v_j$$

$$M_F = \rho_f V_0$$

$$M = \rho V_0$$

$$\Rightarrow u = v \quad \text{if} \quad \rho_f = \rho$$

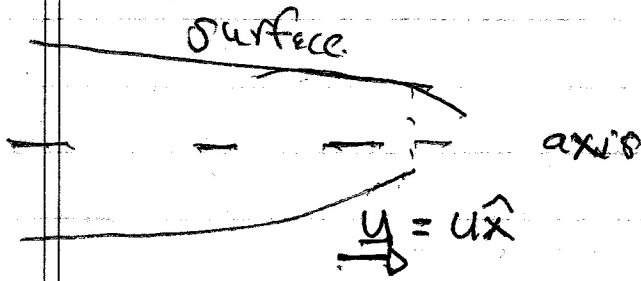
$$u < v \quad \text{if} \quad \rho_f < \rho \quad \rightarrow \text{heavy object legs}$$

$\rho_f \equiv$ fluid density
 $\rho \equiv$ body density

$$u > v \quad \text{if} \quad \rho_f > \rho \quad \rightarrow \text{light object legs}$$

c.) Potential Flow - General Slender Body

- Till now, have considered simple body potential flows, i.e. sphere, cylinder,
 Here consider general body from surface of revolution



- i.e.
- generally axially symmetric slender body
 - slender $\Rightarrow w/L \ll 1$

Now, observe analogy with electrostatics again,

i.e. e.s. $\Rightarrow \phi(x) = \int d^3x' \rho(x') / |\underline{x} - \underline{x}'|$

potential flow ($A \sim uV$)

$$\phi(x) \equiv \frac{1}{4\pi} \int d^3x' (\dot{\rho}(x') / \rho_0) / |\underline{x} - \underline{x}'|$$

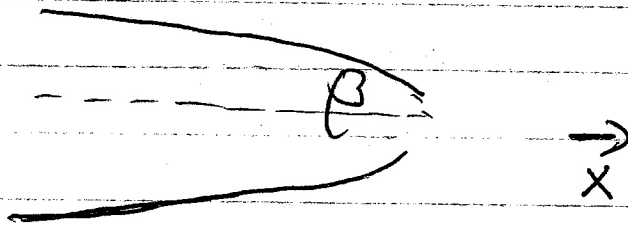
$\frac{\dot{\rho}(x')}{\rho_0} \equiv$ normalized density of fluid flowing across cross-section of body

\rightarrow yields $A \sim V_0 U$ etc.

$$\phi(x) = \frac{1}{4\pi |\underline{x}|^2} \int d^3x' \frac{\dot{\rho}(x')}{\rho_0} \underline{x}' + \text{h.o.t.}$$

\downarrow
dipole term dominates

Flow, - body slender $\rightarrow \frac{W}{L} < 1 \Rightarrow \beta \ll 1$



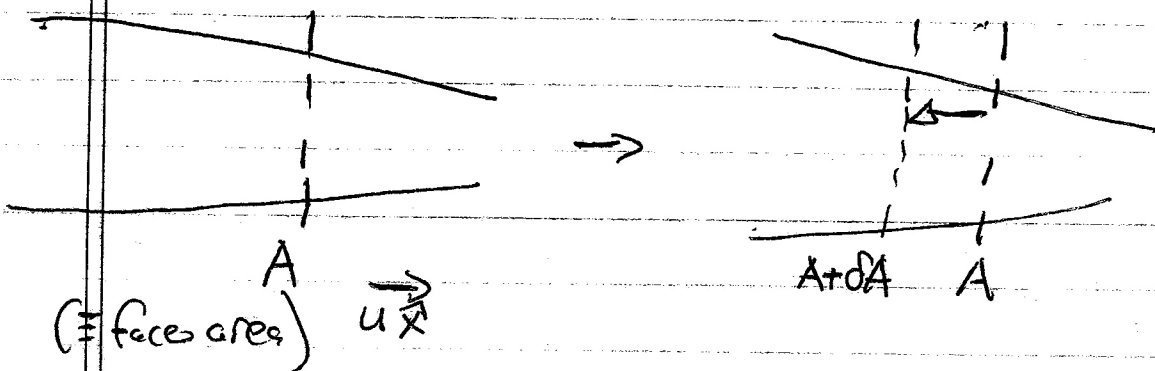
- $\nabla \cdot \underline{V} = 0$ and axial symmetry \Rightarrow

$$\frac{\partial V_x}{\partial x} + \frac{1}{r} \frac{\partial}{\partial r} (r V_r) = 0$$

$$\frac{V_r}{V_x} \sim \frac{\Delta r}{\Delta x} \sim \beta \sim \frac{W}{L} \ll 1$$

\Rightarrow need only consider \hat{x} fluid motion

\therefore to compute dipole moment, need $\rho(x)/\rho_0$ for fluid flow across body



$$\text{Net } \frac{\dot{p}}{\rho_0} = u \left[A + \delta A - A \right] = u \frac{\partial A}{\partial x} dx$$

$$\Rightarrow \rho(x')/\rho_0 = u \frac{\partial A}{\partial x'}$$

$$\therefore \phi(x) = \frac{1}{4\pi r^2} \int dx' x' u \frac{\partial A(x')}{\partial x'}$$

$$= \frac{-u}{4\pi r^2} \int dx' A(x')$$

$$= \frac{-u}{4\pi r^2} V$$

$$V \equiv \text{volume of body} = \int dx' A(x')$$

\Rightarrow yields intuitive result:

$$\phi(x) = \underbrace{-u V_{\text{body}}}_{\text{effective dipole moment for slender body}} / 4\pi r^2$$

effective dipole moment for slender body.