

So

$$\pi' \approx \frac{\partial U_v}{\partial x_k} + \frac{\partial U_c}{\partial x_i}$$

$\underbrace{\hspace{10em}}$
 rate of strain

$$\text{strain} \sim \frac{\partial \underline{\Sigma}}{\partial x_k} + \frac{\partial \underline{\Sigma}_c}{\partial x_i}$$

$\underline{\Sigma} \equiv \text{displacement}$

→ ∞ , most general viscous stress tensor:
shear

$$\pi'_{ijk} = \eta \left(\frac{\partial v_j}{\partial x_k} + \frac{\partial v_k}{\partial x_j} - \frac{2}{3} \delta_{jk} \frac{\partial v_p}{\partial x_p} \right) + \rho \delta_{ijk} \left(\frac{\partial v_p}{\partial x_p} \right)$$


(2/3 ensures vanish for comp)


compressive

$\eta \equiv$ shear viscosity

$\rho \nu \equiv$ kinematic viscosity (diffusion)

$\rho \equiv$ compressional / scaling

η  layers sliding

ζ  compression

is general: Navier-Stokes Eqn. (2nd order contrast Euler)

$$\rho \left[\frac{\partial \underline{v}}{\partial t} + \underline{v} \cdot \nabla \underline{v} \right] = -\nabla p + \mu \nabla^2 \underline{v} + \left(\gamma \frac{\pm 1}{\theta} \right) \nabla (\nabla \cdot \underline{v})$$

$$\nabla \cdot \underline{v} = 0 \Rightarrow$$

$$\left[\frac{\partial \underline{v}}{\partial t} + \underline{v} \cdot \nabla \underline{v} \right] = -\frac{\nabla p}{\rho} + \frac{\mu}{\rho} \nabla^2 \underline{v}$$

~~Reynolds~~

Re = VL/ν ⇒ Reynolds #!

~ ratio of VL to diffusion

→ See notes:

Fluid ~~is~~ F near Maxwellian

$$f = f_{\max} + \delta f$$

where, $\delta f \approx -\frac{v \cdot \nabla f_0}{\gamma}$

$$\pi = \begin{cases} \pi_{\text{ideal}} \rightarrow f = f_{\max} \\ \pi_{\text{ideal}} + \pi_{\text{visc}} \rightarrow f = f_{\max} + \delta f \end{cases}$$

deviation from pure local
Maxwellian induces
viscous stress

- Viscosity reflects difference relaxation
to global Maxwellian.

- Calculate π_{visc} from $\delta f \rightarrow$ notes.

→ will focus on both low and high Reynolds #

→ Vorticity

$$\partial_t \underline{\omega} + \underline{v} \cdot \nabla \underline{\omega} = \underline{\omega} \cdot \nabla \underline{v} - \underline{\omega} \nabla \cdot \underline{v} + \nu \nabla^2 \underline{\omega}$$

$$\partial_t \underline{\omega} = + \nabla \times (\underline{v} \times \underline{\omega}) + \nu \nabla^2 \underline{\omega}$$

for $\nabla \cdot \underline{v} = 0$

$$\partial_t \underline{\omega} + \underline{v} \cdot \nabla \underline{\omega} = \underline{\omega} \cdot \nabla \underline{v} + \nu \nabla^2 \underline{\omega}$$

→ Apart Reynolds:

$$\underline{\pi}_{ij} = \delta_{ij} p + \eta \left(\frac{\partial v_j}{\partial x_i} + \frac{\partial v_i}{\partial x_j} \right)$$

$$(\nabla \cdot \underline{\pi})_i = -\partial_i p + \eta \nabla^2 v_i$$

Some #'s:

H ₂ O	$\nu = .01$	cm ² /sec
air	$\nu = .15$	" "
glycerine	$\nu = 6.8$	" "
Mercury	$\nu = .001$	" "

both { viscous and pressure stress } contribute.

→ Low Re, (Quasi-steady)

$$Re \ll 1,$$

$$\frac{\partial \underline{v}}{\partial t} + \underline{v} \cdot \nabla \underline{v} = - \frac{\nabla p}{\rho} + \nu \nabla^2 \underline{v}$$

$$\nabla \cdot \underline{v} = 0$$

$\nabla p = \mu \nabla^2 \underline{v}$ $\nabla \cdot \underline{v} = 0$
--

Stokes (in)

Hydrodynamics

$$\nabla^2 \underline{\omega} = 0.$$

(2D: $\nabla^2 \nabla^2 \phi = 0$
biharmonic)

IV.B. → lines

→ non-evolutionary

i.e. no ∂_t

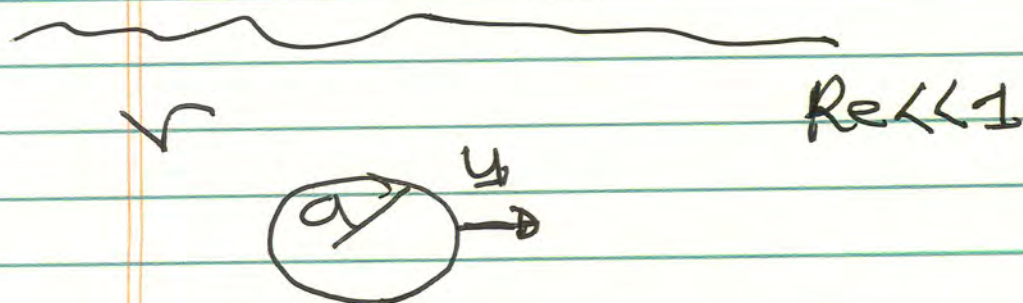
→ more concretely,
reversible

i.e. $\underline{V} \rightarrow -\underline{V}$
 $P \rightarrow C - P$ } recover
 Stokes Eqs.
 C to avoid negative
 pressure

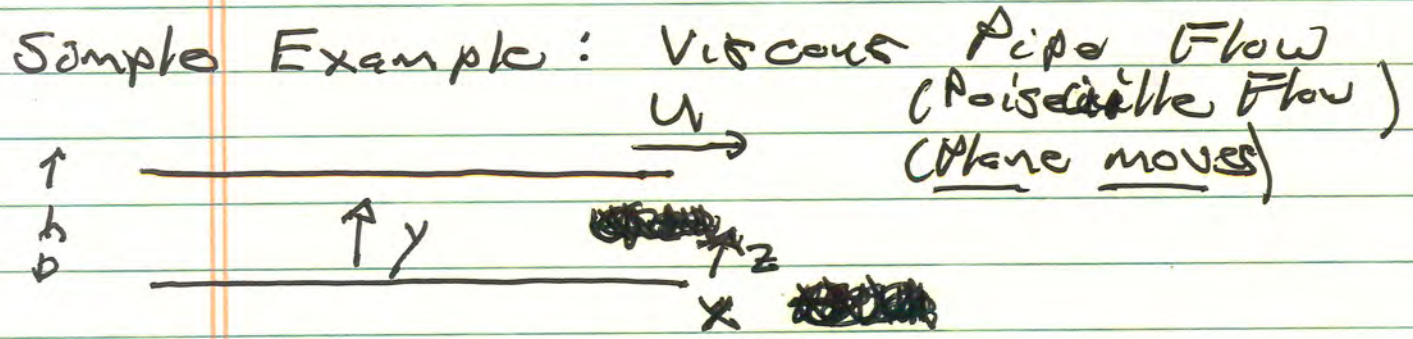
leaves Stokes equations invariant

→ Stokes Drag.

Consider sphere of radius a
 in steady motion of velocity
 \underline{u} in Stokesian fluid.



Calculate force of drag.



$$\frac{\partial \underline{v}}{\partial t} + \underline{v} \cdot \nabla \underline{v} = -\frac{1}{\rho} \nabla p + \nu \nabla^2 \underline{v}$$

steady

$$\underline{v} = v(y) \hat{x}$$

then

$$\nu \nabla^2 v(y) \hat{x} = 0$$

$$\frac{d^2 v}{dy^2} = 0$$

$$\frac{dp}{dy} = 0$$

$$p = p_0$$

so $v = y u / h$

$v = 0$, fixed lower plate

$v = u$, upper

$$\langle v \rangle = \frac{1}{h} \int_0^h v dy = \frac{u}{2}$$

For normal force on plane;

$$\frac{F}{A} = p$$

For tangential force; (on $y=0$ plane)

$$\frac{\tau_{xy}}{\pi_{xy}} \Big|_0 = \eta \frac{\partial u}{\partial y} = \eta \frac{dV}{dy} = \frac{\eta U}{h} \quad (\text{pusher})$$

$$\frac{\tau_{xy}}{\pi_{xy}} \Big|_h = -\frac{\eta U}{h} \quad (\text{opposer})$$

Now consider fixed plates, with pressure gradient along flow:

$\overline{\overline{\Sigma}} \leftarrow DP$ driven flow.

$$\cancel{\frac{\partial p}{\partial t}} + \cancel{v \cdot \nabla} \underline{u} = -\frac{\partial p}{\partial x} + \eta \nabla^2 \underline{u}$$

$$0 = -\frac{\partial p}{\partial x} + \eta \frac{\partial^2 u}{\partial y^2}$$

balance pressure drop by re-distribution of momentum.

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$$\frac{\partial^2 v}{\partial y^2} = \frac{1}{\eta} \frac{\partial p}{\partial x} ; \quad \frac{\partial p}{\partial y} = 0$$

811

→ p indep y

→ p = p(x)

$$\frac{\partial^2 v}{\partial y^2} = \frac{1}{\eta} \frac{\partial p}{\partial x}$$

↓
f(y)
only

↓
f(x)
only

pressure drop

→ momentum input.

where go,
for steady state?

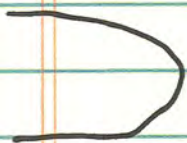
so both const, to be equal.

$$\therefore v = \frac{1}{2\eta} \frac{\partial p}{\partial x} y^2 + ay + b$$

b.c. $v(0) = v(h) = 0$

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$$v = -\frac{1}{2\eta} \frac{dp}{dx} y(y-h)$$



parabolic profile

$$\bar{v} = \frac{-1}{12\eta} h^2 \frac{dp}{dx}$$

↓
pressure drop
 $\Delta P / l$

shear stress:

$$\tau_{xy} \Big|_0 = \eta \frac{dv}{dy} \Big|_{y=0} = \frac{1}{2} h \frac{dp}{dx}$$

Pipe (Circular) → HW.

Example

7.

$$\rightarrow \underline{F_d} = -6\pi\eta a \underline{v}$$

d.e. aka' Aristotle \rightarrow Friction !

$$\rightarrow -6\pi\eta a \underline{v} \quad (\text{not effective inertial})$$

- easily recovered by dimensional analysis (up to 6π)

\rightarrow Key feature:

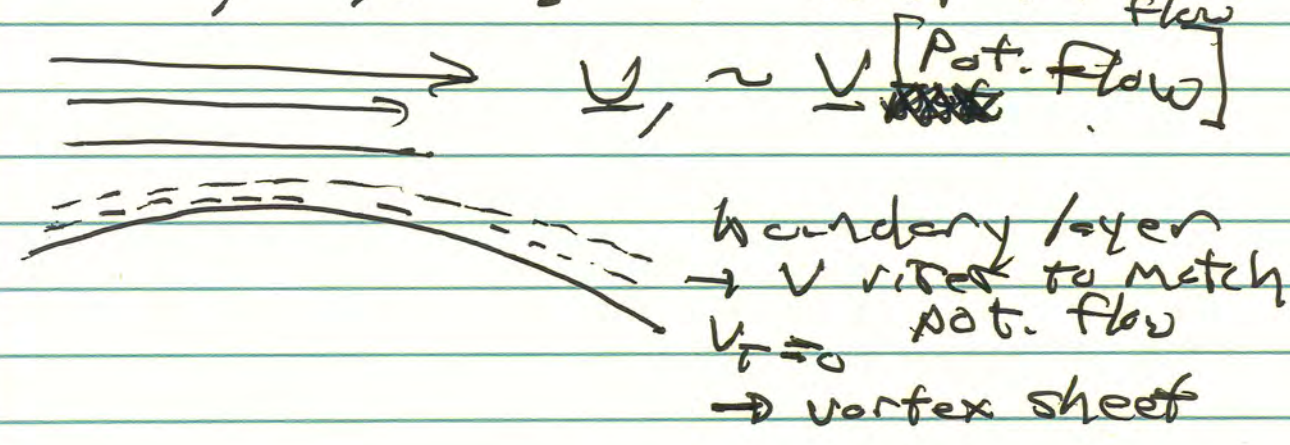
No slip boundary condition

$$\underline{v_n = 0} \quad \underline{v_T = 0}$$

\rightarrow tangential velocity of fluid at surface bounding the fluid vanishes

\rightarrow in addition to $v_n = 0$,

→ boundary layers ! "far field" → potential flow



→ but Stokesian fluid is viscous dominated (i.e. all BL).

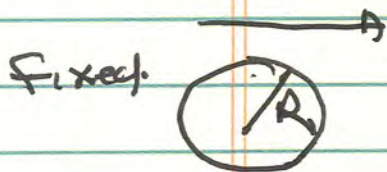
→ Calculating the Drag.

Recall

$$\eta \nabla^2 \underline{v} = \nabla p$$

$$\nabla \cdot \underline{v} = 0$$

$$\eta \nabla^2 \underline{\omega} = 0$$



\underline{v} = flow.

Strategy: Compute flow and pressure fields, then integrated stress on sphere.

~~xxxx~~ $\nabla \cdot (\underline{v} - \underline{u}) = 0$

$$\underline{v} = \underline{u} + \nabla \times \underline{A}$$

axial
 (curl A \rightarrow polar)
 direction

Now,

- A must be axial

- A \sim u, linear

axial: not reversed under $x \rightarrow -x$
 polar \rightarrow reversed under $x \rightarrow -x$

\vec{r}/r
 \downarrow
 scalar
 pot. \downarrow

$$\underline{A} = \left[F'(r) \hat{n} \times \underline{y} \right] \quad \text{general form}$$

$$\nabla \times \underline{A} \rightarrow \underline{y}$$

$$\underline{v} = \nabla \times (\nabla f \times \underline{y}) + \underline{y}$$

$$= \underline{y} + \nabla \times \nabla \times (f \underline{y})$$

\underline{y} const.

Then,

$$\begin{aligned} \nabla \times \underline{v} &= \nabla \times \nabla \times \nabla \times (f \underline{y}) \\ &= (\cancel{\nabla}(\nabla \cdot) - \nabla^2) \nabla \times (f \underline{y}) \\ &= -\nabla^2 (\nabla \times f \underline{y}) \end{aligned}$$

$$\underline{\omega} = -\nabla^2 (\nabla \times f \underline{y})$$

$$\nabla^2 \underline{\omega} = 0$$

\Rightarrow

$$(\nabla^2)^2 \nabla \times (f \underline{y}) = (\nabla^2)^3 (\nabla f \times \underline{y}) = 0$$

$$\begin{aligned}\nabla^2 \underline{\omega} = 0 &= (\nabla^2)^2 (\underline{\nabla F} \times \underline{y}) \\ &= (\nabla^2)^2 \underline{\nabla F} \times \underline{y} = 0\end{aligned}$$

$$\boxed{(\nabla^2)^2 \underline{\nabla F} = 0}$$

harmonic
for scalar.

$$(\nabla^2)^2 f = \text{const.}$$

$$\rightarrow 0$$

as $\underline{v} \rightarrow \underline{y}$, so $\underline{v} - \underline{y} \rightarrow 0$
must vanish at ∞ .

Symmetry $\rightarrow f = f(r)$

$$\nabla^2 \nabla^2 f = \frac{1}{r^2} \frac{d}{dr} \left(r^2 \frac{d}{dr} \right) [\nabla^2 f] = 0$$

$$\nabla^2 f = \frac{2a}{r} + c \quad (\text{Norton})$$

$c \rightarrow 0$ if $\underline{v} \rightarrow \underline{y}$ at ∞ .

$$\boxed{f = ar + b/r}$$

So

$$\underline{V} = \underline{y} + \nabla \times \nabla \times (\underline{Fy})$$

$$F = ar + b/r$$

So

$$\underline{V} = \underline{y} - a \left[\frac{(\underline{y} + \hat{n}(\underline{y} \cdot \hat{n}))}{r} \right] + b \left[\frac{3\hat{n}(\underline{y} \cdot \hat{n}) - \underline{y}}{r^3} \right]$$

Now, $\underline{V} = 0$ at $r = R$,
(+ ~~and~~ \hat{n})

$$-\underline{y} \left(\frac{a}{R} + \frac{b}{R^3} - 1 \right) + \hat{n}(\underline{y} \cdot \hat{n}) \left(\frac{-a}{R} + \frac{3b}{R^3} \right) = 0$$

all \hat{n} \rightarrow each cancel.

$$a = \frac{3}{4} R$$

$$b = \frac{1}{4} R^3$$

$$F = \frac{3}{4} Rr + \frac{1}{4} R^3 / r$$

At last:

$$\begin{aligned}
 v_r &= u \cos \theta \left[1 - \frac{3R}{2r} + \frac{R^3}{2r^3} \right] \\
 v_\theta &= -u \cos \theta \left[1 - \frac{3R}{4r} - \frac{R^3}{4r^3} \right]
 \end{aligned}
 \tag{1}$$

→ velocity field.

Velocity

For stress, need pressure:

$$\underline{\nabla p} = \mu \nabla^2 \underline{v}$$

$$= \mu \nabla^2 \left[\nabla \times \nabla \times (F_y) \right]$$

$$= \mu \nabla^2 \left[\nabla \left[\nabla \cdot (F_y) \right] - \nabla^2 F_y \right]$$

$$(\nabla^2)^2 F = 0$$

$$\Rightarrow \underline{\nabla p} = \underline{\nabla} \left[\mu \nabla^2 \left[\nabla \cdot (F_y) \right] \right]$$

$$= \underline{\nabla} (\mu \nabla \cdot \nabla^2 F)$$

$$\underline{\sigma}_p = \underline{\sigma} (\underline{u} \cdot \underline{\sigma} \nabla^2 F)$$

$$p = \underline{u} \cdot \underline{\sigma} \nabla^2 F + p_0$$

Plug in for F :

$$p = p_0 - \frac{3}{2} \eta \frac{\underline{u} \cdot \underline{\hat{n}}}{r^2} R$$

pressure (2)

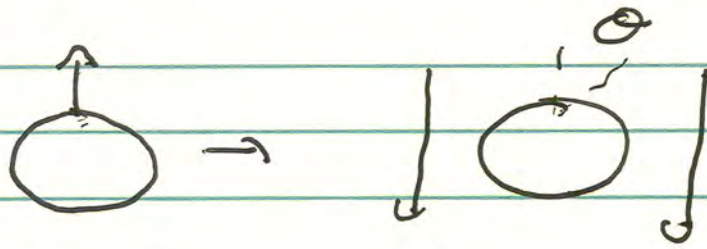
(1), (2) tell story.

For force on sphere:
pressure

$$F_i = \int ds \left(p n_i - \underbrace{\tau_{ik} n_k}_{\text{viscous stress}} \right)$$

($Re \ll 1 \rightarrow$ no Reynolds contribution).

Now



normal

$$F = \int ds \left[\begin{array}{l} -p \cos \theta + \pi'_{rr} \cos \theta \\ \text{pressure head} \quad \text{viscous} \\ -\pi'_{r\theta} \sin \theta \end{array} \right]$$

$$\pi'_{rr} = 2\eta \frac{\partial v_r}{\partial r} \quad \text{normal stress}$$

$$\pi'_{r\theta} = \eta \left(\frac{1}{r} \frac{\partial v_r}{\partial \theta} + \frac{\partial v_\theta}{\partial r} - \frac{v_\theta}{r} \right)$$

$$\frac{\partial}{\partial \theta} \downarrow r$$

(r flux of momentum)

tangential stress

$$\pi'_{rr} \Big|_R = 0$$

$$\pi'_{r\theta} \Big|_R = -\frac{3\eta}{2R} u \sin \theta$$

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$$F_d = \frac{3\mu\gamma}{2R} \int ds = 6\pi\eta R\gamma.$$

More generally,

$$F_i = \eta U_{ij} c_{ij}$$

↓
order 4

symmetric, due to F_{ij} .

Example

→ Scallop Theorem and Related

→ Mechanical Fish



→ water → swims

→ can't stretch → no net motion ↓

Why?

Difference \vec{P} → \vec{v} and thus Re

$$Re = \frac{VL}{\nu}$$

$$Re \ll 1$$

$$Re_{H_2O} > 1$$

Why?

$$\text{Recall: } \begin{cases} \nabla \cdot \underline{v} = 0 \\ \nabla^2 \underline{v} = \underline{\nabla} P \\ \nabla^2 \underline{\omega} = 0 \end{cases}$$

→ Point:

- time does not appear

↓

- equations not evolutionary.

- pattern motion same, whether forward or back in time

Now:

- to swim, need reconfigure body shape to propel self.

- at low Re ,

"

flapping reversible → back

flex undoes flex → no motion

↑ ↓

↓ ↑



i.e. $\nabla \cdot \nabla^2 \underline{v} = \nabla p$

$\nabla \cdot \underline{v} = 0$

then $\underline{V} \rightarrow -\underline{V}$

reversible flow

$\rho \rightarrow \rho - \rho(x)$
Lirred

also solves Stokes eqns.

So $\underline{V}_{\text{Boundary}} = \underline{F}(x)$ Flip
 $\rightarrow \underline{V}$

then if reverse

$\underline{V}_{\text{Boundary}} = -\underline{F}(x)$ Back-Flip

$\rightarrow -\underline{V}$ solution

Back-Flip undoes Flip!
[Reversal boundary conditions \rightarrow reversal flow.]

Scallop thm consequence of "reversible" structure of Stokes equations.

→ Why? Uniqueness Thm.
 ↳ 1 solution, per body config.

— viscous fluid
 (Stokes)

— $\int \underline{V} = \underline{V}_B(\underline{x})$ /
 surface \rightarrow on S
 S

then 1 solution

Proof: (contradiction)

\underline{V}^* s.t. $\underline{V}^* = \underline{V}_B(\underline{x})$ on S

another (hypothetical) solution?
 ρ^*

$\underline{U} = \underline{V} - \underline{V}^* \equiv$ difference flow.

$\underline{P} = \rho^* - \rho$

linear equations \Rightarrow

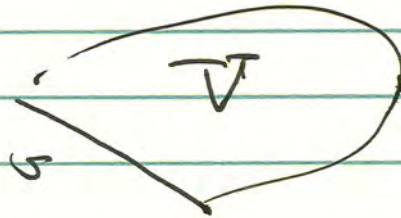
$$\underline{0} = -\underline{\nabla} \mathcal{P} + \eta \nabla^2 \underline{v}$$

$$\underline{0} = -\underline{\nabla} \mathcal{P}^* + \eta \nabla^2 \underline{v}^*$$

so

$$\underline{0} = -\underline{\nabla} \mathcal{P} + \eta \nabla^2 \underline{u}$$

$$\underline{\nabla} \cdot \underline{u} = 0$$



u*

$$\underline{0} = -\underline{u} \cdot \underline{\nabla} \mathcal{P} + \eta \underline{u} : \nabla^2 \underline{u}$$

$$= -\underline{\nabla} \cdot (\mathcal{P} \underline{u}) + \eta (\nabla^2 \underline{u}) \cdot \underline{u}$$

so integrate over flow volume,

$$\underline{0} = - \int_S \mathcal{P} \underline{u} \cdot \underline{\hat{n}} \, dS \quad \left. \begin{array}{l} \underline{u} = \underline{0} \\ \text{on } S \end{array} \right\} \text{(both } \underline{v}, \underline{v}^* \text{ set b.c.)}$$

$$+ \eta \int_V d^3x \left[\frac{\partial}{\partial x_j} \left(\mu_{ij} \frac{\partial u_i}{\partial x_j} \right) - \left(\frac{\partial u_i}{\partial x_j} \right)^2 \right]$$

$$u_i = 0 \text{ on } S.$$

so $\int d^3x \left(\frac{\partial u_i}{\partial x_j} \right)^2 = 0$

→ u_i constant (all i, j)

→ $u_i = 0$ on S → $u_i = 0$

so no other solution.

Corollary → back flow
- reversed flow.

N.B.: Thm. - linearity
- reversibility.

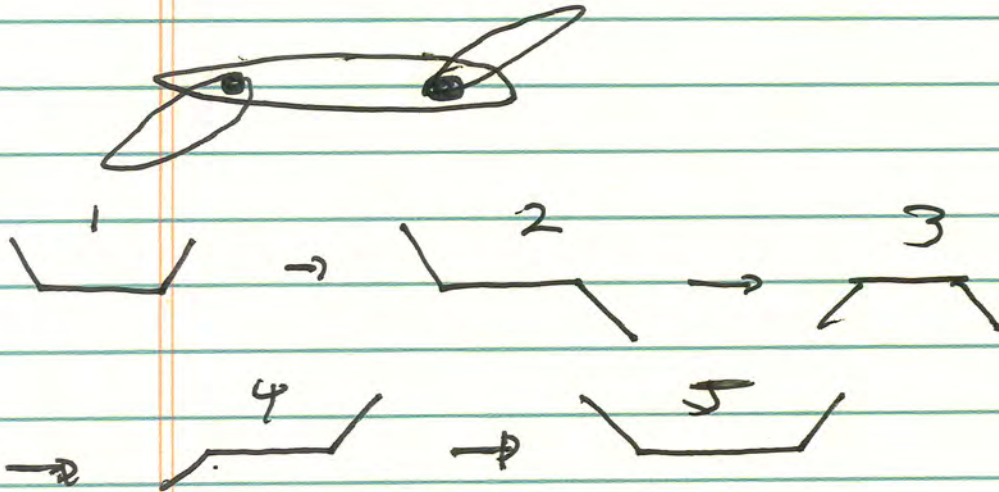
Big Questions:

→ how escape the scallop
thm.?

→ how do micro-organisms swim,
($Re \ll 1$).

Some examples:

- c.) two flappers!

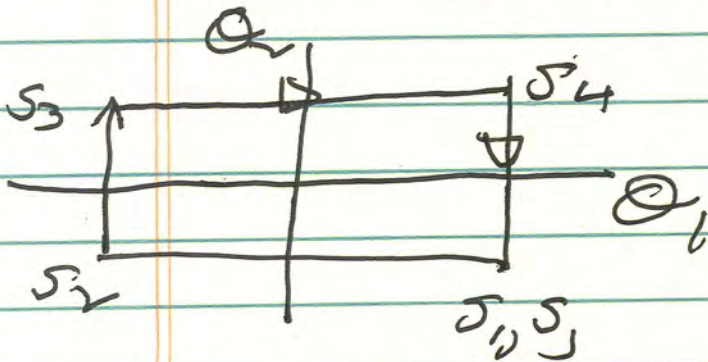


back stroke
diffent forward.

→ animal configuration

→ 2 angles

→ traverses loop



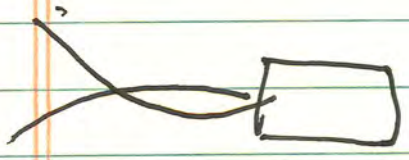
direction

→ clock vs
counter-clock

dc.) Toroidal blob

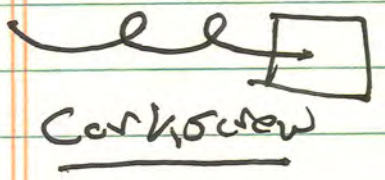


iii.)



flexible ocean (deformed boundary)

iv.)

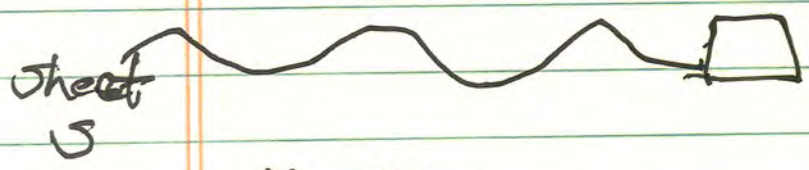


Cork screw

rotated

(common for microorganisms)

Model - Thin Flexible Sheet



$$x = x_s$$
$$y = a \sin(\omega x - \omega t)$$

time and direction

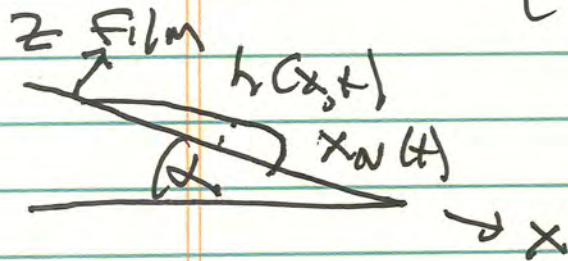
→ y motion of sheet reversible

→ time reversed → wave reversed
in x.!

⇒ steady flow.

→ HW

→ Example Thin film on slope
(Free surface)



Flow down slope

$$0 = -\frac{\partial p}{\partial x} + \mu \frac{\partial^2 u}{\partial z^2} + g \sin \alpha \quad \text{along}$$

\hookrightarrow thin.

$$0 = -\frac{\partial p}{\partial z} - g \cos \alpha \quad \text{perp}$$

Integrating second;

$$p = -\rho g z \cos \alpha + f(x, t)$$

$$z = h(x, t), \quad p = p_0 \quad (\text{Free surface})$$

$$p = \rho g [h(x, t) - z] \cos \alpha + p_0$$

Free surface \rightarrow tangential stress vanishes $\eta \frac{\partial u}{\partial z} = 0$.

Free surface:

$$0 = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \frac{\partial^2 u}{\partial z^2} + g \sin \alpha$$

$$= \rho g \cos \alpha \frac{\partial h}{\partial x} + g \sin \alpha + \nu \frac{\partial^2 u}{\partial z^2}$$

small.

then,

$$\rho \frac{\partial^2 \psi}{\partial z^2} = -\rho g \sin \alpha$$

$$\psi = \frac{\rho g \sin \alpha}{\rho} \left(hz - \frac{z^2}{2} \right)$$

(no slip $z=0$)

$$\frac{\partial w}{\partial z} = \frac{\partial v_z}{\partial z} = -\frac{\partial \psi}{\partial x} = -\frac{\rho g \sin \alpha}{\rho} z \frac{\partial h}{\partial x}$$

$$w = -\frac{\rho g \sin \alpha}{2\rho} \frac{\partial h}{\partial x} z^2$$

Finally: (kinematic free surface condition)

$$w = \frac{\partial h}{\partial t} + u \frac{\partial h}{\partial x} \quad \text{on } z = h(x, t)$$

$$u = \frac{\rho g \sin \alpha}{\rho} \left(hz - \frac{z^2}{2} \right)$$

$$u(h) = \frac{\rho g \sin \alpha}{\rho} \frac{h^2}{2}$$

$$w = \frac{\partial h}{\partial t} + u \frac{\partial h}{\partial x}$$

$$u|_h = \frac{g \sin \alpha}{r} \frac{h^2}{2}$$

$$w = - \frac{g \sin \alpha}{2r} \frac{\partial h}{\partial x} h^2$$

$$= - \frac{g \sin \alpha}{2r} \frac{\partial h}{\partial x} h^2$$

$$- \frac{g \sin \alpha}{2r} \frac{\partial h}{\partial x} h^2 = \frac{\partial h}{\partial t} + \frac{g \sin \alpha}{2r} h^2 \frac{\partial h}{\partial x}$$

80

$$\frac{\partial h}{\partial t} + \frac{g \sin \alpha}{r} h^2 \frac{\partial h}{\partial x} = 0$$

NL
wave.

$$h = f \left(x - \frac{g \sin \alpha}{r} h^2 t \right) \rightarrow \text{similarity !!!}$$

i.e. compare:

$$\frac{\partial \alpha}{\partial t} + c \frac{\partial \alpha}{\partial x} = 0.$$

$$\alpha = \alpha(x - ct)$$

Try: $h = h(x/t^\alpha)$

→

$$-\frac{x}{t^{\alpha+1}} h' + \frac{g \sin \alpha}{r} h^2 \frac{h'}{t^\alpha} = 0$$

$$\frac{h'}{t^\alpha} \left[-\frac{x}{t} + \frac{g \sin \alpha}{r} h^2 \right] = 0$$

$$h \sim x^{1/2} / t^{1/2}$$

→ similarity soln.

How fast does it spread downhill?

note

$$h(x(t)) = A$$

$$\rightarrow x_N \sim t^{2/3}$$